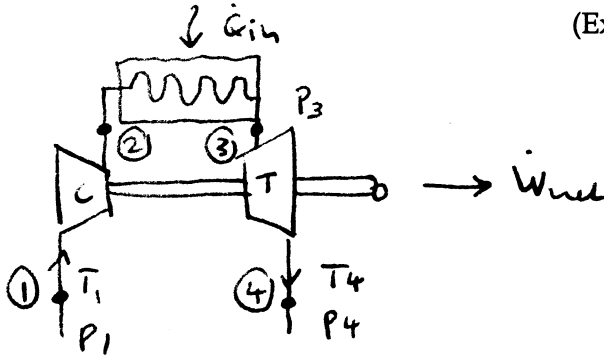


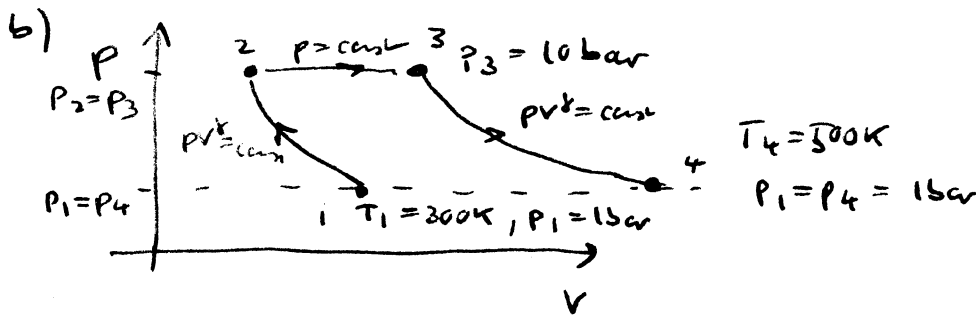
(Extra page for solutions)



Assumptions:

- adiabatic compressor, turbine
- steady operation
- heat addition @ const. pressure
- neglect $\Delta KE, \Delta PE$ effects
- reversible turbomachinery (ideal)
- ideal gas with const. spec. heats

- a) ① → ② : adiabatic reversible compression
 ② → ③ : heat addition at const. pressure (isobaric)
 ③ → ④ : adiabatic reversible expansion



one isobaric process
two adiabatic reversible processes

c) Adiabatic reversible expansion through turbine: $\left(\frac{P_2}{P_4}\right)^{\frac{\gamma-1}{\gamma}} = \frac{T_3}{T_4}$
 $T_3 = T_4 \left(\frac{P_3}{P_4}\right)^{\frac{\gamma-1}{\gamma}}, T_3 = 965 \text{ K}$

d) Isobaric heat addition: $P_2 = P_3$ $P_2 = 10 \text{ bar}$
 Adiabatic reversible compression $T_2 = T_1 \left(\frac{P_2}{P_1}\right)^{\frac{\gamma-1}{\gamma}}, T_2 = 579 \text{ K}$

e)

$$\frac{dE_{cv}}{dt} = \sum \dot{Q} + \sum \dot{w} + \sum \dot{m}(h + \frac{v^2}{2} + gz)$$

$$0 = 0 - \dot{W}_{shft} + \dot{m}(h_1 - h_2 + h_3 - h_4)$$

$$\dot{m} = \frac{\dot{W}_{shft}}{c_p(T_1 - T_2 + T_3 - T_4)} ; \dot{m} = 5.35 \text{ kg/s}$$

f)

1st law CV form: $0 = \dot{W}_{shft}^c + \dot{m}(h_1 - h_2)$; $\dot{W}_{shft}^c = c_p(T_2 - T_1)$
 $\dot{W}_{shft}^c = 280 \text{ kJ/kg}$

g)

1st law CV form: $0 = \dot{Q}_{in} + \dot{m}(h_2 - h_3)$
 $\dot{Q}_{in} = \dot{m} c_p(T_3 - T_2)$ $\dot{Q}_{in} = 2.07 \text{ MW}$